

**ASME B18.2.3.9M-2001**  
[Revision of ANSI/ASME B18.2.3.9M-1984 (R1995)]

# **METRIC HEAVY HEX FLANGE SCREWS**

**AN AMERICAN NATIONAL STANDARD**





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# METRIC HEAVY HEX FLANGE SCREWS

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[Revision of ANSI/ASME B18.2.3.9M-1984 (R1995)]

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## FOREWORD

American National Standards Committee B18 for the standardization of bolts, screws, nuts, rivets, and similar fasteners was organized in March 1922 as Sectional Committee B18 under the aegis of the American Engineering Standards Committee (later the American Standards Association, then the United States of America Standards Institute and, as of October 6, 1969, the American National Standards Institute, Inc.), with the Society of Automotive Engineers and the American Society of Mechanical Engineers as joint sponsors. Subcommittee 2 was subsequently established and charged with the responsibility for technical content of standards covering wrench head bolts and nuts.

At its meeting on December 4, 1974, Committee B18 authorized preparation of a series of standards for metric fasteners. Subcommittee 2 was assigned responsibility for developing standards for metric hex bolts, screws, and nuts.

Following issuance of ANSI B18.2.3.4M-1979, interest developed in the farm equipment, heavy truck, and off-road vehicles industries for a second series of metric hex flange screws having a head widths across flats series equal to those of metric hex cap screws. During the analytical research study to optimize head proportions of hex flange screws (refer to the Foreword of ASME B18.2.3.4M-2000), this larger head series was also investigated, and appropriate values for the various head characteristics were established for screw sizes M10 through M20.

This new design of metric heavy hex flange screws was submitted by the United States of America/Canada to ISO/TC2 at its May 1982 meetings. TC2 accepted the proposal and authorized development of International Organization for Standardization (ISO) standards.

After these ISO decisions, Subcommittee 2 drafted this Standard.

This Standard was approved by letter ballot of the ASME B18 Standards Committee on April 18, 1983, and was subsequently approved by the American Society of Mechanical Engineers and submitted to the American National Standards Institute for designation as an American National Standard. This was granted on March 23, 1984. The Standard was reaffirmed in 1995. This draft revision was prepared based on revisions that were made in the metric hex cap screw standard in December 1998. This Standard was approved by the American National Standards Institute on March 9, 2001. The key revisions include the following:

(a) The heavy hex series is expanded to include sizes M5, M6, and M8 with head dimensions adopted by ISO/DIS 8102.

(b) All head heights ( $K$ ) and wrenching heights ( $K_w$ ) are increased to improve wrenchability.

(c) The position of head-to-shank is changed to maximum material condition (MMC) as described in para. 11 and Tables 2 and 5.

(d) The length tolerances are changed in Table 7 to agree with those of the draft of ASME B18.2.3.4M, Metric Hex Flange Screws.

(e) Straightness is specified at MMC, and a rail gage replaces the sleeve gage as specified in para. 18.

(f) The concavity angle of the conical bearing surface is now  $0.75 \text{ deg} \pm 0.50 \text{ deg}$  to agree with that of ISO and to match B18.2.3.4M. The bearing face runout is to be measured on the actual bearing circle instead of the minimum bearing circle as specified in para. 13.

(g) The transition threads shall have a rounded root contour per para. 20.4. This replaces,

“For screws of Property Class 10.9 and higher strength materials (tensile strength 1040 MPa and higher) the transition threads shall have a rounded root contour no radius of which shall be less than the specified minimum at the root of the full form thread.”

(h) The position of body-to-thread previously required is deleted based on the B18 Subcommittee 2 decision of December 2, 1998.

(i) Appendix A is added to provide a detailed comparison with ISO/DIS 8102, with unpublished changes agreed upon in ISO/TC2/WG2. The current wrenching height formula in ISO 4759-1 would increase the wrenching heights even more. No support exists in North America for increases beyond those included herein.

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The Committee welcomes proposals for revisions to this Standard. Such proposals should be as specific as possible, citing the paragraph number(s), the proposed wording, and a detailed description of the reasons for the proposal, including any pertinent documentation.

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The request for interpretation should be clear and unambiguous. It is further recommended that the inquirer submit his/her request in the following format:

Subject:	Cite the applicable paragraph number(s) and the topic of the inquiry.
Edition:	Cite the applicable edition of the Standard for which the interpretation is being requested.
Question:	Phrase the question as a request for an interpretation of a specific requirement suitable for general understanding and use, not as a request for an approval of a proprietary design or situation. The inquirer may also include any plans or drawings which are necessary to explain the question; however, they should not contain proprietary names or information.

Requests that are not in this format will be rewritten in this format by the Committee prior to being answered, which may inadvertently change the intent of the original request.

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# METRIC HEAVY HEX FLANGE SCREWS

## 1 SCOPE

(a) This Standard covers the complete dimensional and general data for metric series heavy hex flange screws recognized as American National Standard.

(b) The inclusion of dimensional data in this Standard is not intended to imply that all products described are stock production items. Consumers should consult with suppliers concerning availability of products.

## 2 COMPARISON WITH ISO/DIS 8102

(a) Heavy hex flange screws as presented in this standard are harmonized to the extent possible with the draft international standard ISO/DIS 8102, Hexagon Bolts With Flange—Heavy Series, and with revisions agreed to by ISO/TC2/WG2 in May 1989 in Zurich and later that have not been published in a revised draft. If it were published, the dimensional differences between this Standard and ISO/DIS 8102 would be few, relatively minor, and none would affect functional interchangeability of screws manufactured to the requirements of either.

This Standard specifies some requirements that are not included in ISO/DIS 8102. **Dimensional requirements shown in bold type are in addition to, or differ from, ISO/DIS 8102.** The technical differences between this standard and the ISO documents are described in Appendix A.

(b) Letter symbols designating dimensional characteristics are in accord with ISO 225, Fasteners—Bolts, Screws, Studs and Nuts—Symbols and Designations of Dimensions, and with ISO/DIS 8102, except where capitals have been used instead of lowercase letters used in the ISO standards.

## 3 REFERENCED STANDARDS

The following is a list of publications referenced in this Standard. Unless otherwise specified, the standard(s) referenced shall be the most recent issue at the time of order placement.

ASME B1.3M Screw Thread Gaging Systems for Dimensional Acceptability—Inch and Metric Threads (UN, UNR, UNJ, M, and MJ)

ASME B1.13M Metric Screw Threads—M Profile

ASME B18.2.8 Clearance Holes for Inch and Metric Bolts and Screws

ASME B18.12 Glossary of Terms for Mechanical Fasteners

ASME B18.18.1M Inspection and Quality Assurance for General Purpose Fasteners

ASME B18.18.2M Inspection and Quality Assurance for High-Volume Machine Assembly Fasteners

ASME B18.24.1 Part Identifying Number (PIN) Code System Standard for B18 Externally Threaded Fasteners

ASME Y14.5M Dimensioning and Tolerancing

Publisher: American Society of Mechanical Engineers (ASME International), Three Park Avenue, New York, NY 10016-5990; Order Department: 22 Law Drive, Box 2300, Fairfield, NJ 07007

ASTM F 468M Nonferrous Bolts, Hex Cap Screws, and Stands for General Use (Metric)

ASTM F 568M Carbon and Alloy Steel Externally Threaded Metric Fasteners

ASTM F 738M Stainless Steel Metric Bolts, Screws, and Studs

ASTM F 788/F 788M Surface Discontinuities of Bolts, Screws and Studs, Inch and Metric Series

Publisher: American Society for Testing and Materials (ASTM), 100 Barr Harbor Drive, West Conshohocken, PA 19428

ISO 225 Fasteners—Bolts, Screws, Studs and Nuts—Symbols and Designations of Dimensions<sup>1</sup>

ISO 3508 Thread Runouts for Fasteners With Thread in Accordance With ISO 261 and ISO 262<sup>1</sup>

ISO 4759-1 Tolerances for Fasteners—Part 1: Bolts, Screws and Nuts With Thread Diameters 1.6 to 150 mm and Product Grades A, B and C<sup>1</sup>

ISO/DIS 8102 Hexagon Bolts With Flange—Heavy Series<sup>1</sup>

<sup>1</sup> May also be obtained from American National Standards Institute, 11 West 42nd Street, New York, NY 10036.



Publisher: International Organization for Standardization (ISO), 1 rue de Varembe, Case Postale 56, CH-1211, Genève 20, Switzerland/Suisse

SAE J1199 Mechanical and Material Requirements for Metric Externally Threaded Steel Fasteners

Publisher: Society of Automotive Engineers (SAE), 400 Commonwealth Drive, Warrendale, PA 15096

## 4 TERMINOLOGY

For definitions of terms relating to fasteners or features thereof used in this standard, refer to ASME B18.12.

## 5 DIMENSIONS

(a) All dimensions in this standard are given in millimeters (mm), and apply before any coating, unless stated otherwise.

(b) Symbols specifying geometric characteristics are in accord with ASME Y14.5M.

## 6 TOP OF HEAD

The top of head shall be either full form or indented at manufacturer's option, and shall be either chamfered or rounded. The diameter of the chamfer circle or start of rounding shall be equal to the maximum width across flats,  $S$  maximum, within a tolerance of  $-15\%$ . If the top of the head is indented, the periphery may be rounded.

## 7 HEAD HEIGHT

The head height,  $K$ , is the distance, parallel to the axis of the screw, from the plane of the bearing circle to the top of the head, not including any raised markings. See para. 22.

## 8 WRENCHING HEIGHT

The wrenching height,  $K_w$ , is the distance at a corner of the hexagon from the junction of hex head with the flange to the last plane of full-formed hexagon (i.e., the plane closest to the top of the head at which the width across corners,  $E$ , of the hexagon is within its specified limits).

## 9 CORNER FILL

The rounding due to lack of fill at the six corners of the head shall be reasonably uniform.

## 10 GAGING OF HEAVY HEX FLANGE HEAD

See Table 1. The head shall be gaged using two ring gages, A and B, to demonstrate the coincidental acceptability of wrenching height, corner fill, and width across corners. Gage A shall be placed over the head and shall seat on the flange. Gage B shall be placed on the top of the head normal to the screw axis. The two gages shall not be in contact.

## 11 POSITION OF HEAD

At maximum material condition, the axis of the hexagon of the head shall be within a positional tolerance zone of the diameter specified in Table 2 with respect to the axis of the shank over a distance under the head equal to the nominal screw diameter,  $D$ . The datum shall be as close to the head as practicable, but within  $0.5D$  from the head, and shall be either wholly plain body or wholly the thread pitch diameter, not including the thread runout or the underhead fillet.

## 12 FLANGE

The top surface of the flange shall be conical or slightly rounded (convex). Radius,  $R_2$ , applies both at the corners and at the flats of the hexagon. The contour of edge at flange periphery, between the maximum flange diameter,  $D_c$  maximum, and the minimum bearing circle diameter,  $D_w$  minimum, shall be optional provided that the minimum flange edge thickness,  $C$  minimum, is maintained at the minimum bearing circle diameter,  $D_w$  minimum.

## 13 BEARING SURFACE

The bearing surface shall be conical,  $0.75 \pm 0.50$  deg concave from the plane formed by the bearing circle diameter. The plane formed by the bearing circle shall be perpendicular to the axis of the shank, over a length under the head equal to the nominal screw diameter,  $D$ , within the circular runout as specified in Table 2. The measurement of bearing face runout shall be made at the actual bearing circle (i.e., at the line of highest points on any radial line, e.g., by use of straight edge anvil). The datum shall be as close to the head as practical, and shall be either wholly plain body or

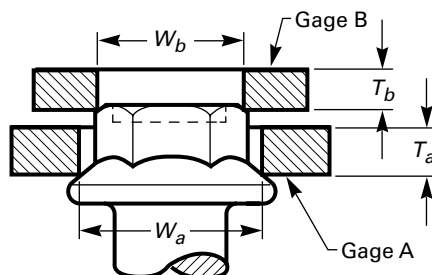


TABLE 1 GAGING OF HEAVY HEX FLANGE HEADS

Nominal Screw Diameter and Thread Pitch	Gage A				Gage B		
	Inside Diameter, $W_a$		Thickness, $T_a$		Inside Diameter, $W_b$		Thickness, $T_b$
	Max.		Max.		Max.		Min.
	Min. [Note (1)]		Min. [Note (2)]		Min. [Note (3)]		Min.
M5 × 0.8	9.25	9.24	2.40	2.39	8.55	8.54	3.0
M6 × 1	11.56	11.55	2.70	2.69	10.79	10.78	3.0
M8 × 1.25	15.02	15.01	3.60	3.59	14.07	14.06	4.0
M10 × 1.5	17.33	17.32	4.60	4.59	16.31	16.30	4.0
M12 × 1.75	20.79	20.78	5.20	5.19	19.67	19.66	5.0
M14 × 2	24.26	24.25	6.30	6.29	22.57	22.56	5.0
M16 × 2	27.72	27.71	7.10	7.09	25.93	25.92	6.0
M20 × 2.5	34.65	34.64	8.80	8.79	32.65	32.64	6.0

## GENERAL NOTES:

- (a) Dimensions are in millimeters.  
 (b) Refer to para. 10.

## NOTES:

- (1)  $W_a$  min. equals theoretical maximum width across corners,  $E$  max.  
 (2)  $T_a$  max. equals minimum wrenching height,  $K_w$  min.  
 (3)  $W_b$  max. equals minimum width across corners,  $E$  min., -0.01 mm.

TABLE 2 TOLERANCE ZONES

Nominal Screw Diameter and Thread Pitch	Position of Head-to-Shank Tolerance Zone Diameter at MMC	Circular Runout of Bearing Circle to Shank FIM [Note (1)]
M5 × 0.8	0.44	0.17
M6 × 1	0.44	0.21
M8 × 1.25	0.54	0.27
M10 × 1.5	0.54	0.34
M12 × 1.75	0.54	0.42
M14 × 2	0.66	0.48
M16 × 2	0.66	0.56
M20 × 2.5	0.66	0.70

GENERAL NOTE: Dimensions are in millimeters.

## NOTE:

- (1) Circular runout of bearing circle to shank is based on 1 deg and the minimum bearing circle diameter,  $D_w$  min.

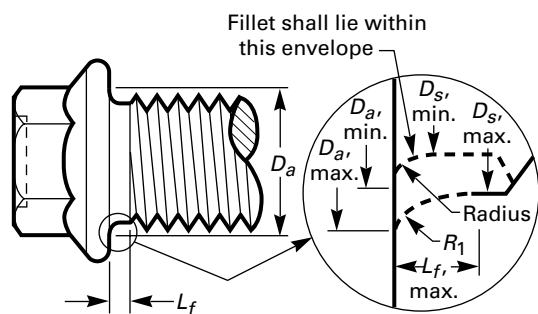
wholly thread pitch diameter, not including the thread runout or the underhead fillet.

## 14 FILLET

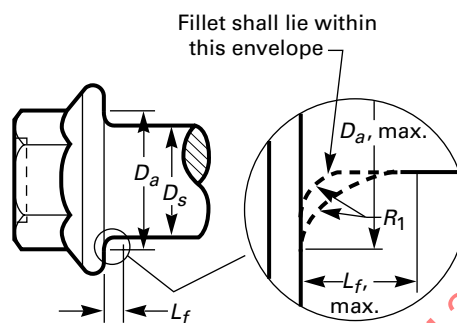
The fillet configuration at the junction of the head and shank shall conform to either Type F, as shown in Table 3, which also specifies limits, or Type U, as shown in Table 4, which also specifies limits, at the option of the manufacturer unless the fillet type is specified by the purchaser. The fillet shall be a smooth and continuous curve fairing smoothly into the underhead bearing surface and the shank within the limits specified. For Type F, no radius in the fillet contour shall be less than  $R_1$  minimum specified in Table 3.

## 15 BODY DIAMETER

The diameter of the body,  $D_s$ , on screws that are not threaded full length shall be within the limits



(a) Detail for Short Screws



(b) Detail for Long Screws

TABLE 3 DIMENSIONS OF TYPE F UNDERHEAD FILLETS

Nominal Screw Diameter and Thread Pitch	Fillet Transition Diameter, $D_a$		Body Diameter, $D_s$ [Note (2)]	Fillet Length, $L_f$		Fillet Radius, $R_1$
	For Short [Note (1)] and Long Screws	For Short [Note (1)] Screws	For Short [Note (1)] Screws	For Long Screws	For Short [Note (1)] Screws	For Short [Note (1)] and Long Screws
	Max.	Min.	Min.	Max.	Max.	Min.
M5 × 0.8	5.7	5.1	4.36	1.4	0.7	0.2
M6 × 1	6.8	6.2	5.21	1.6	0.9	0.25
M8 × 1.25	9.2	8.3	7.04	2.1	1.1	0.4
M10 × 1.5	11.2	10.2	8.86	2.1	1.2	0.4
M12 × 1.75	13.7	12.2	10.68	2.1	1.3	0.6
M14 × 2	15.7	14.1	12.50	2.1	1.4	0.6
M16 × 2	17.7	16.5	14.50	3.2	1.6	0.6
M20 × 2.5	22.4	20.7	18.16	4.2	2.1	0.8

## GENERAL NOTES:

(a) Dimensions are in millimeters.

(b) Type F was formerly named Style A.

## NOTES:

(1) Short screws are screws that are threaded full length.

(2) Values of  $D_s$  for long screws and  $D_s$  max. for short screws are specified in Table 5.

specified in Table 5, unless the purchaser specifies screws with reduced diameter body. For screws threaded full length, the diameter of the unthreaded shank under the head shall neither exceed the maximum body diameter,  $D_s$  maximum, specified in Table 5 nor be less than the minimum body diameter,  $D_s$  minimum, specified in Table 3 or 4.

Screws of nominal lengths equal to or greater than the shortest nominal lengths specified in Table 6 may be obtained with reduced diameter body, if so specified. Where reduced diameter body (or "Type R") is specified, the body diameter,  $D_2$ , shall be within the limits specified in Table 6. The screw shall have a shoulder under the head. The diameter,  $D_s$ , and length,  $L_2$ , of the shoulder shall be as specified in Table 6.

## 16 LENGTH

The length of the screw shall be measured parallel to the axis of the screw from the plane formed by the underhead bearing circle to the extreme end of the shank. Tolerances for screw lengths are specified in Table 7.

## 17 POINTS

The end of the screw shall be chamfered or rounded from a diameter equal to or slightly less than the thread root diameter. The length of the point to the first full-formed thread at major diameter, as determined by the distance the point enters into a cylindrical NOT GO

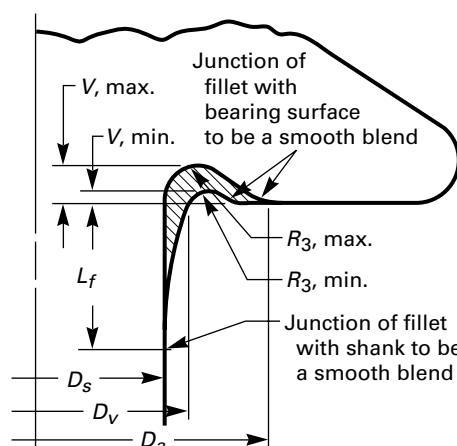


TABLE 4 DIMENSIONS OF TYPE U UNDERHEAD FILLETS

Nominal Screw Diameter and Thread Pitch	Undercut Diameter, $D_u$ [Note (1)]	Fillet Diameter, $D_v$ [Note (2)]	Body Diameter For Short Screws, $D_s$ [Notes (3) and (4)]	Fillet Length, $L_f$	Undercut Radius, $R_3$		Undercut Depth, $V$	
	Max.	Max.	Min.	Max.	Max.	Min.	Max.	Min.
M5 × 0.8	6.2	5.5	<b>4.36</b>	1.4	0.25	0.10	0.15	0.05
M6 × 1	7.5	6.6	<b>5.21</b>	1.6	0.26	0.11	0.20	0.05
M8 × 1.25	10.0	8.8	<b>7.04</b>	2.1	0.36	0.16	0.25	0.10
M10 × 1.5	12.5	10.8	<b>8.86</b>	2.1	0.45	0.20	0.30	0.15
M12 × 1.75	15.2	12.8	<b>10.68</b>	2.1	0.54	0.24	0.35	0.15
M14 × 2	17.7	14.8	<b>12.50</b>	2.1	0.63	0.28	0.45	0.20
M16 × 2	20.5	17.2	<b>14.50</b>	3.2	0.72	0.32	0.50	0.25
M20 × 2.5	25.7	21.6	<b>18.16</b>	4.2	0.90	0.40	0.65	0.30

## GENERAL NOTES:

(a) Dimensions are in millimeters.

(b) Type U was formerly named Style B.

## NOTES:

(1)  $D_u$  was formerly  $D_u$ .(2)  $D_v$  was formerly  $D_a$ .(3) Values of  $D_s$  for long screws and  $D_s$  max. for short screws are specified in Table 5.

(4) Short screws are screws that are threaded full length.

major diameter ring gage, shall not exceed  $U$  maximum specified in Table 8. The end of the screw shall be reasonably square with the axis of the screw, and where pointed blanks are used, the slight rim or cup resulting from roll threading shall be permissible. **At the manufacturer's option, the end of the screw may have a rounded point of radius,  $R_e$ , as specified in Table 8.**

## 18 STRAIGHTNESS

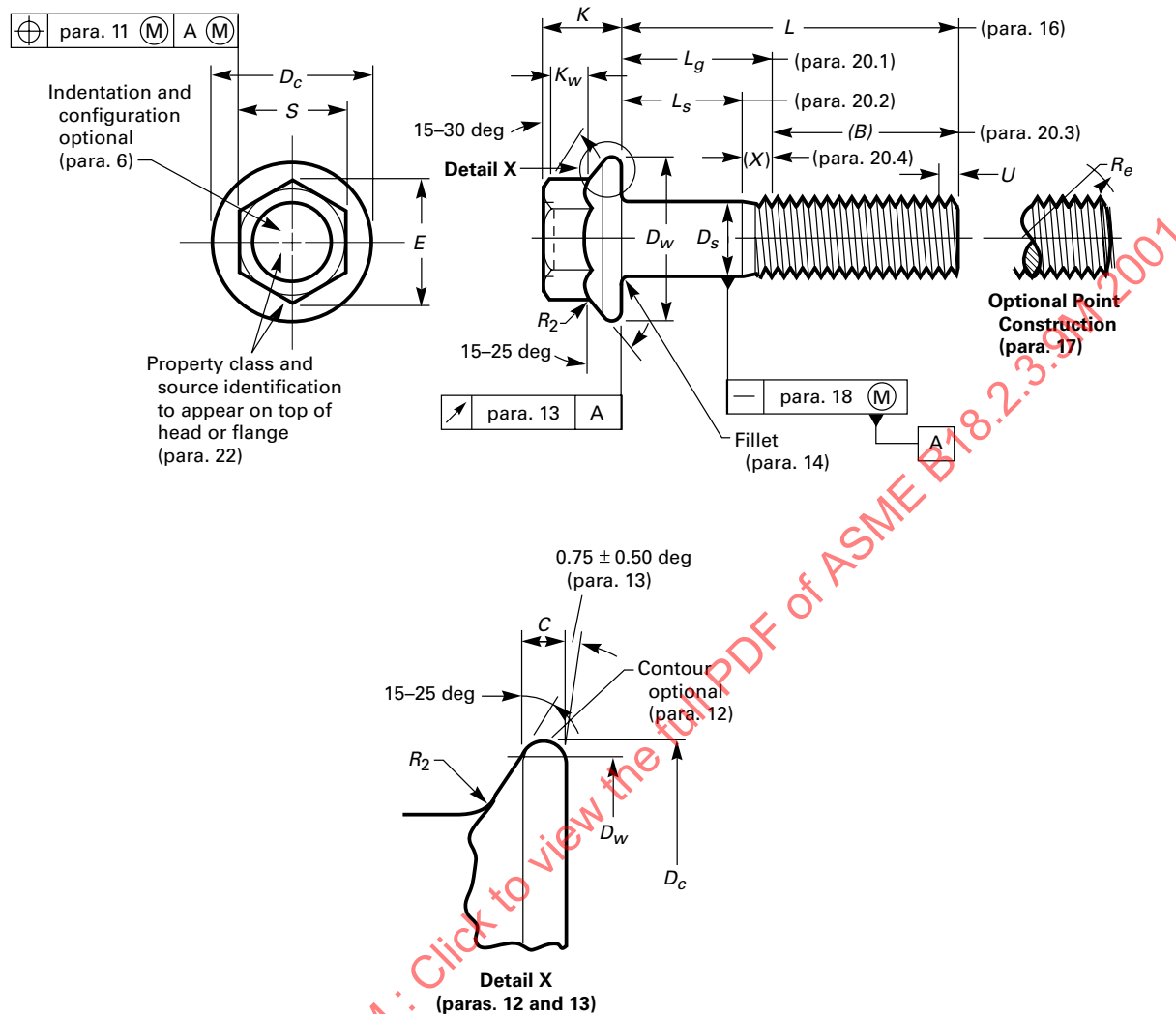
At maximum material condition, the derived median line of the screw body and thread major diameter shall

be within a straightness tolerance zone of a diameter equal to **0.006 times length, expressed as a two-place decimal**. A gage and gaging procedure for checking screw straightness are given in Appendix B.

## 19 THREADS

## 19.1 Thread Series and Tolerance Class

Screw threads shall be standard coarse pitch external metric screw threads with tolerance Class 6g conforming to ASME B1.13M, unless otherwise specified by the



GENERAL NOTE: Dimensions are in millimeters.



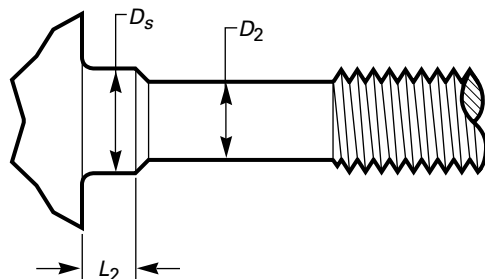


TABLE 6 DIMENSIONS OF REDUCED BODY DIAMETER (TYPE R)

Nominal Screw Diameter and Thread Pitch	Shortest Nominal Screw Length, $L$	Reduced Body Diameter, $D_2$		Shoulder Diameter, $D_s$ [Note (1)]		Shoulder Length, $L_2$ [Note (1)]	
		Max.	Min.	Max.	Min.	Max.	Min.
M5 $\times$ 0.8	30	4.54	4.36	5.00	4.82	3.5	2.5
M6 $\times$ 1	35	5.39	5.21	6.00	5.82	4.0	3.0
M8 $\times$ 1.25	40	7.26	7.04	8.00	7.78	5.0	4.0
M10 $\times$ 1.5	45	9.08	8.86	10.00	9.78	6.0	5.0
M12 $\times$ 1.75	50	10.95	10.68	12.00	11.73	7.0	6.0
M14 $\times$ 2	55	12.77	12.50	14.00	13.73	8.0	7.0
M16 $\times$ 2	60	14.77	14.50	16.00	15.73	9.0	8.0
M20 $\times$ 2.5	70	18.49	18.16	20.00	19.67	11.0	10.0

GENERAL NOTE: Dimensions are in millimeters.

NOTE:

(1) Shoulder is mandatory. See para. 15.

TABLE 7 LENGTH TOLERANCES

Nominal Length, mm	Length Tolerance, mm
Over 6 to 10	$\pm 0.29$
Over 10 to 18	$\pm 0.35$
Over 18 to 30	$\pm 0.42$
Over 30 to 50	$\pm 0.5$
Over 50 to 80	$\pm 0.6$
Over 80 to 120	$\pm 0.7$
Over 120 to 180	$\pm 0.8$
Over 180 to 250	$\pm 0.925$
Over 250 to 300	$\pm 1.05$

purchaser. For screws with additive finish, size limits for tolerance Class 6g apply before plating or coating, and the thread profile after coating is subject to acceptance using a 6h GO gage and tolerance Class 6g thread gage for either minimum material, LO or NOT GO.

## 19.2 Thread Gaging

Unless otherwise specified, dimensional acceptability of screw threads shall be determined based on System 21, ASME B1.3M.

TABLE 8 DIMENSIONS OF POINTS

Nominal Screw Diameter and Thread Pitch	Approximate Point Radius, $R_p$	Max. Point Length, $U$
M5 $\times$ 0.8	7.0	1.60
M6 $\times$ 1	8.4	2.00
M8 $\times$ 1.25	11.2	2.50
M10 $\times$ 1.5	14.0	3.00
M12 $\times$ 1.75	16.8	3.50
M14 $\times$ 2	19.6	4.00
M16 $\times$ 2	22.4	4.00
M20 $\times$ 2.5	28.0	5.00

GENERAL NOTES:

- (a) Dimensions are in millimeters.  
 (b)  $R_p$  equals 1.4 times the nominal screw diameter, and conforms with ISO 4753.  
 (c)  $U$  max. equals two times the thread pitch.

## 20 THREAD LENGTH

The length of thread on screws is controlled by the maximum grip length,  $L_g$  maximum, and the minimum body length,  $L_s$  minimum, as set forth in paras. 20.1 through 20.4.

**TABLE 9 MAXIMUM GRIP LENGTHS,  $L_g$ , AND MINIMUM BODY LENGTHS,  $L_s$** 

Nominal Length, $L$	Nominal Diameter and Thread Pitch															
	M5 × 0.8		M6 × 1		M8 × 1.25		M10 × 1.5		M12 × 1.75		M14 × 2		M16 × 2		M20 × 2.5	
	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.	$L_g$ Max.	$L_s$ Min.
8	1.2	...														
10	2.4	...	1.5	...												
12	2.4	...	3.0	...	1.9	...										
16	2.4	...	3.0	...	4.0	...	2.2	...								
20	2.4	...	3.0	...	4.0	...	4.5	...	2.6	...	3.0	...				
25	9.0	5.0	3.0	...	4.0	...	4.5	...	5.3	...	3.0	...	3.0	...		
30	14.0	10.0	12.0	7.0	4.0	...	4.5	...	5.3	...	6.0	...	3.0	...	3.8	...
35	19.0	15.0	17.0	12.0	13.0	6.75	4.5	...	5.3	...	6.0	...	6.0	...	3.8	...
40	24.0	20.0	22.0	17.0	18.0	11.75	14.0	6.5	5.3	...	6.0	...	6.0	...	7.5	...
45	29.0	25.0	27.0	22.0	23.0	16.75	19.0	11.5	15.0	6.25	6.0	...	6.0	...	7.5	...
50	34.0	30.0	32.0	27.0	28.0	21.75	24.0	16.5	20.0	11.25	16.0	6.0	6.0	...	7.5	...
55			37.0	32.0	33.0	26.75	29.0	21.5	25.0	16.25	21.0	11.0	17.0	7.0	7.5	...
60			42.0	37.0	38.0	31.75	34.0	26.5	30.0	21.25	26.0	16.0	22.0	12.0	7.5	...
65					43.0	36.75	39.0	31.5	35.0	26.25	31.0	21.0	27.0	17.0	19.0	6.5
70					48.0	41.75	44.0	36.5	40.0	31.25	36.0	26.0	32.0	22.0	24.0	11.5
80					58.0	51.75	54.0	46.5	50.0	41.25	46.0	36.0	42.0	32.0	34.0	21.5
90							64.0	56.5	60.0	51.25	56.0	46.0	52.0	42.0	44.0	31.5
100							74.0	66.5	70.0	61.25	66.0	56.0	62.0	52.0	54.0	41.5
110									80.0	71.25	76.0	66.0	72.0	62.0	64.0	51.5
120									90.0	81.25	86.0	76.0	82.0	72.0	74.0	61.5
130											90.0	80.0	86.0	76.0	78.0	65.5
140											100.0	90.0	96.0	86.0	88.0	75.5
150													106.0	96.0	98.0	85.5
160													116.0	106.0	108.0	95.5
180															128.0	115.5
200															148.0	135.5

**GENERAL NOTES:**

- (a) Dimensions are in millimeters.  
 (b) Diameter-length combinations between the thin stepped lines are recommended.  
 (c) Screws with lengths above the solid thick stepped lines are threaded full length. See Table 10.  
 (d) For screws with longer lengths,  $L_g$  and  $L_s$  values shall be computed from formulas as given in para. 20.

**20.1 Grip Length,  $L_g$** 

The grip length,  $L_g$ , is the distance, measured parallel to the axis of the screw, from the plane of the bearing circle to the face of a noncounterbored, noncountersunk GO thread ring gage assembled by hand as far as the thread will permit. For standard diameter-length combinations of screws, the values for  $L_g$  maximum are specified in Table 9. For diameter-length combina-

tions not listed in Table 9, the maximum grip length for long screws that are not threaded full length is equal to the nominal screw length,  $L$ , minus the reference thread length,  $(B)$ , as specified in Table 10:  $L_g$  maximum =  $L$  nominal -  $(B)$ .

For short screws of nominal lengths,  $L$ , that are shorter than the lengths specified in Table 10 for screws threaded full length,  $L_g$  maximum =  $A$  maximum as specified in Table 10.

TABLE 10 THREAD LENGTHS

Noninal Screw Diameter and Thread Pitch	Thread Length, (B, Ref.)				Screws Threaded Full Length					
	Screw Length, L, ≤125	Screw Length, L, >125 and ≤200	Screw Length, L, >200	Transition Thread Length (X, Ref.)	Screw Length, L	Unthreaded Length Under Head, A	Screw Length, L		Unthreaded Length Under Head, A	
						Under	Max.	At Least	Under	Max.
M5 × 0.8	16	22	35	4.0	10	1.2	10	25	2.4	
M6 × 1	18	24	37	5.0	12	1.5	12	30	3.0	
M8 × 1.25	22	28	41	6.25	16	1.9	16	35	4.0	
M10 × 1.5	26	32	45	7.5	20	2.2	20	40	4.5	
M12 × 1.75	30	36	49	8.75	24	2.6	24	45	5.3	
M14 × 2	34	40	53	10.0	28	3.0	28	50	6.0	
M16 × 2	38	44	57	10.0	32	3.0	32	55	6.0	
M20 × 2.5	46	52	65	12.5	40	3.8	40	65	7.5	

GENERAL NOTE: Dimensions are in millimeters.

## 20.2 Body Length, $L_s$

Body length,  $L_s$ , on long screws that are not threaded full length is the distance, parallel to the axis of the screw, from the plane of the bearing circle to the last scratch of thread or top of the extrusion angle, whichever is closer to the head. For standard diameter-length combinations of screws, the values for  $L_s$  minimum are specified in Table 9. For diameter-length combinations not listed in Table 9, the minimum body length on long screws that are not threaded full length is equal to the maximum grip length, as determined above, minus the reference transition thread length, (X), as specified in Table 10:  $L_s$  minimum =  $L_g$  maximum - (X).

## 20.3 Thread Length, (B)

The thread length, (B), specified in Table 10, is a reference dimension intended for calculation purposes only, and is the distance, parallel to the axis of the screw, from the extreme end of the screw to the last complete (full-form) thread.

## 20.4 Transition Thread Length, (X)

The transition thread length, (X), specified in Table 10, is a reference dimension intended for calculation purposes only. It includes the length of incomplete threads and tolerances on grip length and body length. The transition from full-form thread to incomplete thread shall be smooth and uniform. **The major diameter of the incomplete threads shall not exceed the actual major diameter of the complete (full-form) threads.**

The transition threads shall have a rounded root contour.

## 21 MATERIALS AND MECHANICAL PROPERTIES

### 21.1 Steel

Unless otherwise specified, steel screws shall conform to the requirements for Property Class 9.8 (8.8 for size M20) or Property Class 10.9 as specified in ASTM F 568M, or SAE J1199.

### 21.2 Corrosion-Resistant Steels

Unless otherwise specified, corrosion-resistant steel screws shall conform to the requirements of ASTM F 738M.

### 21.3 Nonferrous Metals

Unless otherwise specified, nonferrous screws shall conform to the requirements of ASTM F 468M.

## 22 IDENTIFICATION SYMBOLS

Markings shall be raised or recessed on the top of the head **or raised on the top of the flange** unless otherwise specified by the purchaser. Markings shall be legible to the unaided eye with the exception of corrective lenses. **When raised, markings shall project not less than 0.1 mm for M14 and smaller screws, and 0.3 mm for M16 and M20 screws, above the**

top surface of the head or flange; and total head height (head plus markings) shall not exceed the specified maximum head height,  $K$  maximum, plus 0.1 mm for M5 and M6 screws, 0.2 mm for M8 and M10 screws, 0.3 mm for M12 and M14 screws, and 0.4 mm for M16 and M20 screws.

## 22.1 Property Class Symbols

Each screw shall be marked in accordance with the requirements of the applicable specification for its material and mechanical properties.

## 22.2 Source Symbols

Each screw shall be marked to identify its source (manufacturer or private label distributor).

## 23 FINISH

Unless otherwise specified, screws shall be supplied with a natural (as processed) finish, unplated or uncoated, in a clean condition, and lightly oiled.

## 24 WORKMANSHIP

Screws shall be free from surface imperfections such as burrs, seams, laps, loose scale, and other surface irregularities that could affect serviceability and shall conform to ASTM F 788/F 788M.

## 25 INSPECTION AND QUALITY ASSURANCE

Unless otherwise specified, acceptability of screws shall be determined in accordance with ASME B18.18.1M.

## 26 DIMENSIONAL CONFORMANCE

Products shall conform to the specified dimensions. Unless otherwise specified, the following provisions shall apply for inspection of dimensional characteristics:

(a) Unless otherwise specified, the following designated dimensional characteristics shall be inspected to the inspection levels shown according to ASME B18.18.2M.

Characteristic	Inspection Level
Thread acceptability	C
Body diameter, $D_s$	C
Gaging of hex flange head	C
Grip length, $L_g$ max.	C
Screw length, $L$	C
Visual inspection [Note (1)]	C

### NOTE:

(1) Visual inspection shall include property class marking, source marking, fillet, and workmanship.

If verifiable in-process inspection is used, inspection sample sizes and reporting shall be in accordance with the applicable ASME, ASTM, or SAE quality system consensus standard.

(b) For nondesignated dimensional characteristics, the provisions of ASME B18.18.1M shall apply. Should a non-designated dimension be determined to be outside its specified limits, it shall be deemed conforming to this Standard if the user, who is the installer, accepts the dimension, based on form, fit, and function considerations.

## 27 CLEARANCE HOLES

The recommended sizes of clearance holes in material to be assembled using heavy hex flange screws are the normal series given in ASME B18.2.8.

## 28 DESIGNATION

(a) Heavy hex flange screws shall be designated by the following data, preferably in the sequence shown: product name and designation of the standard, nominal diameter and thread pitch, nominal length, steel property class or material identification, and protective coating, if required.

### EXAMPLES:

- (1) Heavy hex flange screw B18.2.3.9M, M10  $\times$  1.5  $\times$  50, Class 9.8, zinc plated per ASTM F 871M and ASTM B 633 Fe/Zn 5 Type II.
- (2) Heavy hex flange screw B18.2.3.9M, M16  $\times$  2  $\times$  100, Class 10.9, phosphate/oil, ASTM F 1137 Grade I.
- (3) Heavy hex flange screw B18.2.3.9M, reduced diameter body, fillet Type U, M16  $\times$  2  $\times$  80, Class 10.9 oiled.
- (4) Heavy hex flange screw B18.2.3.9M, M16  $\times$  1  $\times$  30, corrosion resistant steel A1-70 ASTM F 738M.

NOTE: It is common practice in ISO standards to omit thread pitch from the nominal size designation when screw threads are the metric coarse thread series, e.g., M10 is M10  $\times$  1.5.

(b) For a recommended part identifying number (PIN) system for metric heavy hex flange screws, see ASME B18.24.1.

## NONMANDATORY APPENDIX A COMPARISON WITH ISO STANDARDS

This Appendix describes the technical differences between ASME B18.2.3.9M-2001 and ISO/DIS 8102, with unpublished changes agreed upon in ISO/TC2/WG2, May 1989 and later, and related ISO standards.

### A1 PRODUCT NAMES *BOLT* AND *SCREW*

Of the products named *hex flange screw* per B18.2.3.9M, those that are threaded full length would be named *hexagon screw with flange* per ISO 1891, but those that are not threaded full length are named *hexagon bolt with flange* per ISO 1891 and ISO/DIS 8102.

### A2 MAXIMUM WIDTH ACROSS CORNERS

B18.2.3.9M Table 1 maximum width across corners is not specified in ISO 8102. Instead, ISO/DIS 4759-1: May 1998 would control the shape of the hexagon by zero positional tolerance at maximum material condition, which is not specified in B18.2.3.9M.

### A3 GAGING OF HEX FLANGE HEAD

ISO/DIS 8102 would specify a gage C for gaging flange thickness, which is not included in B18.2.3.9M.

### A4 POSITION OF HEAD

B18.2.3.9M para. 11 and Table 2 specify positional tolerance at maximum material condition, as in ISO/DIS 4759-1: May 1998, instead of regardless of feature size, as in ISO 4759/I-1978. ISO/DIS 4759-1: May 1998 also specifies position of the flange diameter with respect to the shank, which is not specified in B18.2.3.9M.

### A5 BEARING SURFACE

B18.2.3.9M Table 2 values for circular runout of the bearing circle are greater than those in ISO 4759/I-1978 or ISO/DIS 4759-1: May 1998. ISO/DIS 4759-

1: May 1998 specifies straightness of radial lines on the bearing surface, which is not specified in B18.2.3.9M.

### A6 FILLETS

B18.2.3.9M Table 3 minimum fillet transition diameters and shorter maximum fillet lengths for screws threaded full length are not specified in ISO/DIS 8102. For Type U fillet, the radius of the fillet extension,  $r_4$  reference in ISO/DIS 8102, is not specified in B18.2.3.9M.

### A7 BODY DIAMETER FOR SCREWS THREADED FULL LENGTH

B18.2.3.9M Tables 3 and 4 minimum diameters of unthreaded shank for screws threaded full length are not specified in ISO/DIS 8102.

### A8 REDUCED DIAMETER BODY

B18.2.3.9M Table 6 maximum and minimum diameters of reduced body differ from ISO 8102 " $d_2$  is approximately equal to the pitch diameter (rolling diameter)." The limits specified in B18.2.3.9M Table 6 were proposed to ISO/TC2/WG2 in May 1995, but ISO/TC2/WG2 was not willing to specify limits for the rolling diameter.

### A9 POINT

B18.2.3.9M para. 17 and Table 5 optional rounded point is not in ISO/DIS 8102, but Table 8 agrees with ISO 4753.

### A10 STRAIGHTNESS

B18.2.3.9M specifies straightness at maximum material condition, as in ISO/DIS 4759-1: May 1998, instead of regardless of feature size, as in ISO 4759/I-1978. The formula in B18.2.3.9M results in straightness tolerance zone diameters that are smaller for lengths 12 mm and shorter, and larger for lengths 16 mm and



longer, than those in ISO 4759/I-1978 or ISO/DIS 4759-1: May 1998.

#### **A11 GRIP LENGTH FOR SCREWS THREADED FULL LENGTH**

B18.2.3.9M Tables 9 and 10 maximum unthreaded lengths under the head of screws shorter than  $2D$  differ from ISO 3508.

#### **A12 TRANSITION THREADS**

B18.2.3.9M para. 20.4 specification on root contour of transition threads is not in ISO/DIS 8102.

#### **A13 POSITION OF BODY-TO-THREAD**

Coaxiality or position of the body with respect to the thread, specified in ISO 4759/I or ISO/DIS 4759-1, is not specified in B18.2.3.9M.

#### **A14 MARKING**

B18.2.3.9M para. 22 allows markings on the top of the flange but not on the side of the head, as alternatives to markings on the top of the head, while ISO 898-1 allows markings on the side of the head but not on the top of the flange. B18.2.3.9M para. 22 dimensions of head markings are not specified in ISO/DIS 8102 or ISO 898-1.

#### **A15 INSPECTION AND QUALITY ASSURANCE**

B18.2.3.9M paras. 25 and 26 inspection and quality assurance provisions differ from those in ISO/DIS 8102.